

CHANGE OF DISTILLER PERFORMANCE WITH FOULING

Frank Bodendieck and Klaus Genthner

University of Bremen, Germany

Keywords: Steam Consumption, Evaporator, macrofouling, microfouling, Tube Fouling, Alkaline fouling

Contents

1. Introduction
 2. Relationship Between Fouling and Steam Consumption
 3. Performance of the Evaporator Under Different Tube Fouling Conditions
 - 3.1. Design Specifications and Heat Balances of an MSF Plant
 4. The Effect of Various Fouling Conditions on Various Plant Sections
 - 4.1. Performance Ratio
 - 4.2. Overall Heat Transfer Coefficient and Heat Flux per Stage
 - 4.3. The Effect of Various Fouling Conditions on Temperature Distribution in the MSF Plant
 - 4.4. The Effect of Various Fouling Conditions on Flow Changes of the MSF Plant
 - 4.5. The Effect of Various Fouling Conditions on the Distillate Purity of an MSF Plant
- Glossary
Bibliography and Suggestions for further study

Summary

The performance of an MSF distiller is affected by several factors, the most important of these being the fouling of the heat exchange surfaces. Fouling occurs in the heat exchanger tubes in all parts of the MSF plant. This will considerably affect the performance ratio and the production rate of the distiller.

The effects of various fouling factors within each stage in the evaporator performance of the MSF plant, the change in the overall heat transfer coefficient, the stage temperature distributions, the effect of the distillate purity, and further hydraulic and thermal effects on the MSF plant will be shown in the following article.

1. Introduction

The performance of a multistage flash (MSF) distiller is affected by several factors, the most important of these being fouling on the heat exchange surfaces. This will considerably affect the performance ratio and the production rate of the distiller. Other factors which influence the performance of the distiller, for example the salinity and alkalinity of seawater, will only have a marginal effect on the evaporator.

The term fouling refers to the deposition of material on a heat transfer surface, usually resulting in an increase in the resistance to the heat transfer and a subsequent loss of thermal exchange capacity of the heat transfer equipment. Fouling occurs in the heat

exchanger tubes in all parts of the MSF plant: brine heater, heat recovery section, and heat reject section.

The rejection section suffers from macrofouling (e.g. algae and mussels), slimy microfouling, water borne particles, and corrosion products from the tubes. Debris filter, chlorination, and on-load ball cleaning have proved to be successful countermeasures. The brine heater and recovery section mainly suffer from scaling by calcium carbonate and magnesium hydroxide formation when seawater is heated. This kind of fouling increases with increasing temperatures in the MSF plant and is often at a maximum in the brine heater. Possible countermeasures are the use of acid treatment with decarbonization, additives, acid shots, off-load acid cleaning, and on-load sponge ball cleaning.

Tube fouling in the brine heater and in the recovery section causes a higher steam consumption and influences the product flow rate.

2. Relationship Between Fouling and Steam Consumption

When a completely clean MSF plant becomes fouled, the steam consumption will rise and the plant performance will change. Figure 1 shows the steam consumption and product flow of a $4500 \text{ m}^3 \text{ day}^{-1}$ MSF plant as a function of the plant fouling.

A fall in the heat transfer coefficient due to fouling will increase the heat input to the plant and, hence, increase steam consumption in the brine heater. The brine heater fouling is compensated for by increasing the steam temperature and pressure. Increased steam consumption also means increased heat input to the reject section, which then has to be compensated for by an increased cooling water flow. Somewhere above the design point, the fouling effects cannot be compensated for any further. Any additional fouling will then result in a decreased product flow with constant heat consumption.

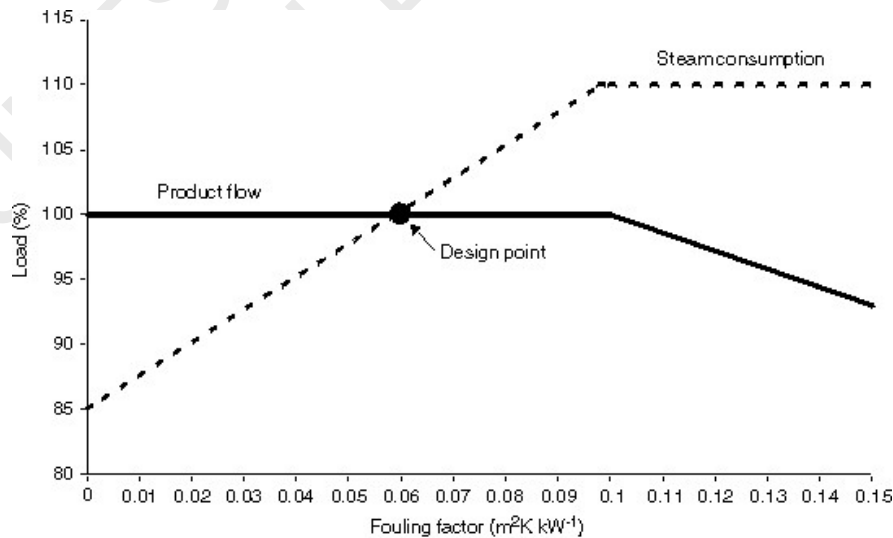


Figure 1. Relationship between steam consumption and fouling factor (Wade 1978).

3. Performance of the Evaporator Under Different Tube Fouling Conditions

Current practice in MSF distiller design is to assign constant fouling factors to the brine heater, the heat recovery section condensers, and to the rejection section pre-heaters. However, this is not in conformity with the actual fouling situation for several reasons.

- (a) Alkaline fouling depends largely on the tube-side wall temperature. Thus, fouling will rapidly decrease with falling stage temperatures.
- (b) Sponge ball cleaning removes soft scale, thus producing an asymptotic fouling resistance, which probably reaches the design values only in the top temperature stages.
- (c) Non-condensable gases can produce, by blanketing the tube sections, a heat transfer resistance of the same order as the design fouling resistance.

With the ball cleaning system in operation, the performance ratios and, hence, the MSF evaporator performance in continuous operation are normally higher than the design specification.

3.1. Design Specifications and Heat Balances of an MSF Plant

Generally, only the overall heat transfer coefficients and fouling factors of the brine heater, heat recovery section, and heat rejection section are considered for the design of the desalination distiller. In reality the fouling factors vary within the different stages of a plant. The consideration of the varying fouling factors within the stages is very important for the determination of an optimum operation point. A desalination unit designed according to this aspect could be operated more efficiently, because the scale-inhibiting tools such as additive dosing and ball cleaning could be designed and operated to suit the plant specifications.

The effects of various fouling factors on the MSF evaporator performance within the stages of a typical MSF plant will be shown in the following sections. Table 1 shows the essential design data of the typical MSF recycle plant.

Thermal and hydraulic data	Nominal operation: summer	Nominal operation: winter
Seawater temperature (°C)	32.0	18.0
Top brine temperature (°C)	105.0	98.0
Capacity (m ³ day ⁻¹)	30 000	30 000
Brine recycle flow (t h ⁻¹)	11 945	11 163
Maximum weir load (t h ⁻¹ m ⁻¹)	948	886
Concentration of seawater (g kg ⁻¹)	45	45
Concentration factor	1.4	1.4
Performance ratio	8.3	8.2
Mechanical design data	Recovery	Rejection
Number of stages	17	3
Number of tubes per stage	2832	2580
Tube internal diameter/thickness (mm)	27.8/1.1	29/1.1

Tube length (m)	≈13.9	≈13.9
Heat transfer area (m ²)	63 120	10 582
Chamber length (m)	St. 1-11: 3.80 St. 12-17: 4.70	4.70
Chamber width (m)	14.0	14.0
Tube material	CuNi 70/30	Titanium
Design fouling factor (m ² K W ⁻¹)	0.00012	0.00015

Table 1 MSF plant design data

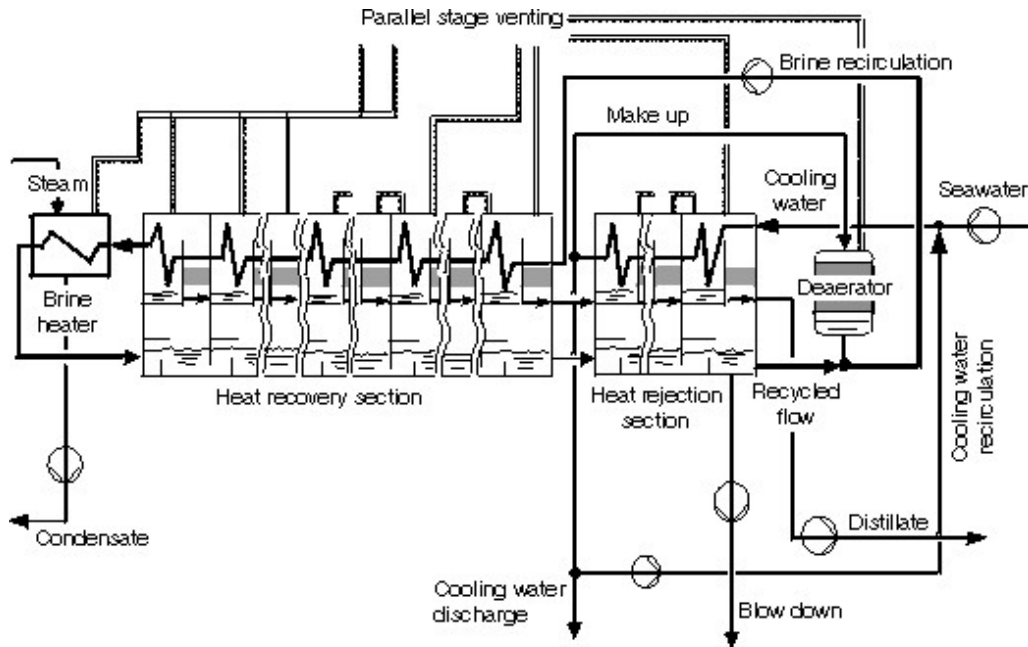


Figure 2. Simplified flow diagram of an MSF recycle plant.

The design data are typical for large MSF plants in operation today in the Middle East, except that here a single tier design was considered. The design of an MSF distiller requires detailed analysis of heat and mass balances, as well as plant cost factors. Figure 2 shows the flow diagram of an MSF recycle process. In general the whole MSF plant can be described by the following energy balances. The specific enthalpy h is a function of the temperature T and the salt concentration ξ of the corresponding mass flow.

(a) Overall heat balance for an MSF recycle plant:

$$\dot{m}_C h(T_S, \xi_S) + \dot{Q}_{BH} = (\dot{m}_C - \dot{m}_{MU}) h(T_{MU}, \xi_S) + \dot{m}_D h(T_D, \xi_D) + \dot{m}_{BD} \cdot h(T_{BD}, \xi_{BD}) \quad (1)$$

Heat balance for the rejection section of the MSF plant:

$$\dot{m}_{D_{NREC}} h(T_{D_{NREC}}, \xi_{D_{NREC}}) + \dot{m}_{NREC} h(T_{NREC}, \xi_{NREC}) + \dot{m}_C h(T_S, \xi_S) = \dot{m}_C h(T_{MU}, \xi_S) + \dot{m}_D h(T_D, \xi_D) + (\dot{m}_{BD} + \dot{m}_R) h(T_{BD}, \xi_{BD}) \quad (2)$$

Heat balance for the recovery section of the MSF plant:

$$\dot{Q}_{BH} + \dot{m}_{BR} h(T_{BR}, \xi_{BR}) = \dot{m}_{D_{NREC}} h(T_{D_{NREC}}, \xi_{D_{NREC}}) + \dot{m}_{NREC} h(T_{NREC}, \xi_{NREC}) \quad (3)$$

Heat balance for the mixing point:

$$\dot{m}_R h(T_{BD}, \xi_{BD}) + \dot{m}_{MU} h(T_{MU}, \xi_S) = \dot{m}_{BR} h(T_{BR}, \xi_{BR}) \quad (4)$$

-
-
-

TO ACCESS ALL THE 18 PAGES OF THIS CHAPTER,
Visit: <http://www.desware.net/DESWARE-SampleAllChapter.aspx>

Bibliography and Suggestions for further study

Abdul Quddus, (2002), *Effect of hydrodynamics on the deposition of CaSO4 Scale on stainless steel*, Desalination, 142, 57-63.

Alahmad M. and Abdul Aleem F.(1993), *Scale Formation and Fouling Problems Effect on The Performance of MSF and RO Desalination Plants in Saudi Arabia*, Desalination, 93, 287-310,.

Al-Gobaisi, D. M. K., (1994), *A Quarter Century Of Seawater Desalination By Large Multistage Flash Plants In Abu Dhabi*, Desalination, 99, 483-509

Al-Sofi M (1985) *Thermal performance of 10x5.2 MIGD MSF plants, Al Jobail Phase II*. Desalination and Water Re-use. Proceedings of Second World Congress, Bermuda, Vol. 2, 1985 (Ed. M Balaban), pp. 357-371. International Desalination Association (IDA), Topsfield, USA.

Al-Sofi, M. AK., El-Sayed, E. F., Imam, M., Mustafa G. M., Hamada, T., Haseba, S. and Goto T., (1995), *Heat Transfer Measurement as a Criterion for Performance Evaluation of Scale Inhibition in MSF plants*, 7th IDA World Congress, Abu Dhabi, UAE, 3, 191.

Babcock (1994) *Optimisation for Maintenance and Operation of Desalination Plants in Abu Dhabi*. Deutsche BABCOCK AG, Deutsche Babcock Energie- und Umwelttechnik AG, Oberhausen, Germany.

Böhmer H (1992) On-load tube cleaning systems and debris filters for avoidance of micro- and macrofouling in MSF-desalination plants. *DESAL*. Proceedings of Arabian Gulf Regional Water

Dalvi, AG. I., Kither, M.N.M., Al-Sulami, S.A., Sahul, K. and Al-Rasheed, (1999), *Effect Of Various Forms Of Iron In Recycle Brine On The Performance Of Scale Control Additives In MSF Desalination Plant*, WSTA 4th Gulf Water Conference, Bahrain, Vol. 2., 663-677.

Desalination Symposium, Al Ain, United Arab Emirates, 1992, Vol. 1 (Ed. M Balaban), pp. 325-335. Water and Electricity Department, Abu Dhabi, UAE.

- Ebrahim A. Hawaidi and Iqbal M. Mujtaba (2010), *Sensitivity of Brine Heater Fouling on Optimization of Operation Parameters of MSF Desalination Process using gPROMS*, 20th European Symposium on Computer Aided Process Engineering – ESCAPE20, S. Pierucci and G. Buzzi Ferraris (Editors)
- Eimer K (1982) Energetic aspects of the fouling of MSF desalination plants with and without the use of on load ball cleaning system. *Desalination* **40**, 363-627.
- El-Dahshan, M.E., (2001), *Corrosion and Scaling Problems Presents in Some Desalination Plants in Abu Dhabi*", *Desalination*, 138, 371-377, .
- El-Dessouky H T and Khalifa T A (1985) Scale formation and 1st effect on the performance of once through MSF plant. *Desalination and Water Re-use*. Proceedings of Second World Congress, Bermuda, 1985, Vol. 2 (Ed. M Balaban), pp. 199-217. International Desalination Association (IDA), Topsfield, USA.
- Georg S (1996) *Ermittlung des Energiebedarfs und der Betriebsdaten von Kreiselpumpenanlagen in mehrstufigen Entspannungsverdampfern der Meerwasser-entsalzung*. M.S. Thesis, Department of Technical Thermodynamics, Heat and Mass Transfer, Bremen University, Bremen.
- Heitmann H G (1990) *Saline Water Processing*, 332 pp. Weinheim: VCH Verlagsgesellschaft.
- Hömig H E (1978) *Seawater and Seawater Distillation*, 202 pp. Essen: Vulkan-Verlag
- Khan A H (1986) *Desalination Processes and Multistage Flash Distillation Practice*, 596 pp. Amsterdam: Elsevier.
- Linnikov, Oleg D.(2003), *About seed concentration for prevention of scale* , *Desalination*, 157, 235-240,.
- Mohammad Abdul-Kareem Al-Sofi (1999), *European Conference on Desalination and the Environment*, Las Palmas, Gran Canaria, Proc., Vol 3, pp. 61-76.
- Mubarak A., (1998) *A Kinetic Model for Scale Formation in MSF desalination Plants. Effect of Antiscalants* , *Desalination* ,120, 33-39.
- Nawaf Naif Al-Mutairi (master degree thesis, 2007),*Fouling Studies and Control in Heat Exchangers* ,King Saud University,College Of Engineering Chemical Engineering Department
- Osman A. Hamed, Khalid B. Mardouf, Adnan Al-Omran (2007),*Impact Of Interruption Of Antiscalant Dosing Or Cleaning Balls Circulation During MSF Plant Operation* *Desalination*, Volume 208, Issues 1-3, Pages 192-203
- Seifert A (1988) *Das Inertgasproblem und Verlusteffekte in Entspannungsverdampfern für die Meerwasserentsalzung*. Ph.D. Thesis, Department of Technical Thermodynamics, Heat and Mass Transfer, Bremen University, Bremen.
- Shams El-Din, A. M. and Rizk, A. M., (1994), *Brine And Scale Chemistry In MSF Distillers*, *Desalination*, 99, 73-111.
- Shams El-Din, A.M. and Rizk, A.M., (1988), *A 700-Day Experiment with Belgerd EV – 2000 as anti-scale agent in MSF distillers*, *Desalination*, 69, 147 – 160.
- Shams El-Din, A.M. and Rizk, A.M., (1989), *The Problem Of Alkaline Scale Formation From A Study On Arabian Gulf Water*, *Desalination*, 71, 313-324.
- Wade N M (1978) A practical comparison between multistage flash and falling film evaporator processes. *Fresh Water from the Sea* (Proceedings of the Sixth International Symposium, Athens, 1978), Vol. 1 (eds A Delyannis and E Delyannis), pp. 327-336. European Federation of Chemical Engineering, Athens, Greece.